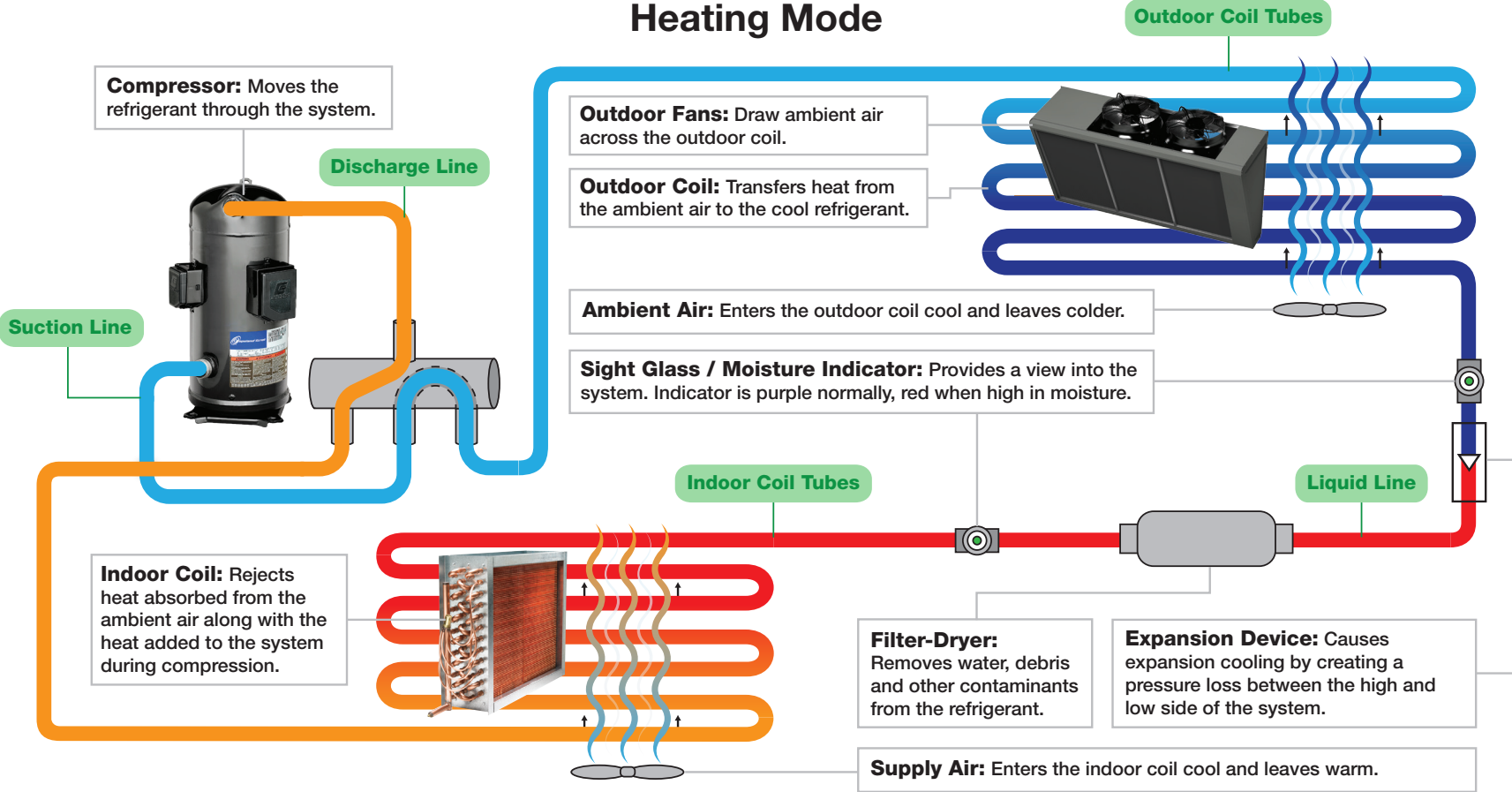


# Air-Source Heat Pump Refrigeration System

## Heating Mode

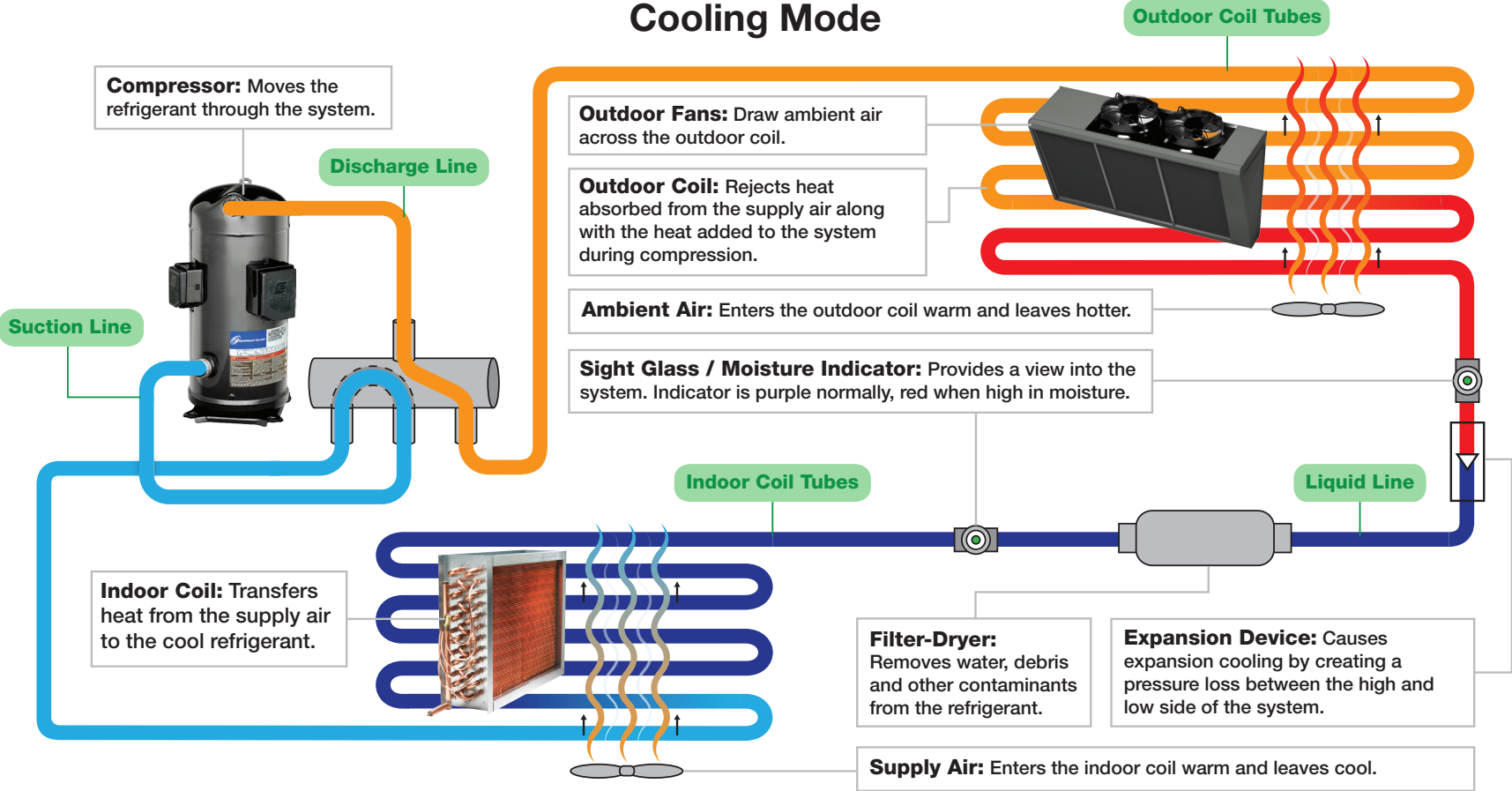


SYSTEM TEMPERATURES AND PRESSURES			
Refrigerant State	T, °F	R-410A, psig	Superheat/Subcool
Low Pressure Saturated	27	91	
Low Pressure Superheated Vapor	37	91	Superheat = $T - T_{sat} = 37^\circ - 27^\circ = 10^\circ\text{F}$
High Pressure Gas	145	319	
High Pressure Saturated	100	319	
High Pressure Liquid	90	319	Subcool = $T_{sat} - T = 100^\circ - 90^\circ = 10^\circ\text{F}$

LOAD CALCULATIONS	
Condenser (Indoor Coil)	$Q_{out} \text{ (Btu/hr)} = 1.08 \times \text{SCFM} \times \Delta \text{ Temperature}$
Evaporator (Outdoor Coil)	$Q_{in} \text{ (Btu/hr)} = 4.5 \times \text{SCFM} \times \Delta \text{ Enthalpy}$

# Air-Source Heat Pump Refrigeration System

## Cooling Mode



SYSTEM TEMPERATURES AND PRESSURES			
Refrigerant State	T, °F	R-410A, psig	Superheat/Subcool
Low Pressure Saturated	45	131	
Low Pressure Superheated Vapor	55	131	Superheat = $T - T_{sat} = 55^\circ - 45^\circ = 10^\circ\text{F}$
High Pressure Gas	160	393	
High Pressure Saturated	115	393	
High Pressure Liquid	105	393	Subcool = $T_{sat} - T = 115^\circ - 105^\circ = 10^\circ\text{F}$

LOAD CALCULATIONS	
Condenser (Outdoor Coil)	$Q_{out} \text{ (Btu/hr)} = 1.08 \times \text{SCFM} \times \Delta \text{ Temperature}$
Evaporator (Indoor Coil)	$Q_{out} \text{ (Btu/hr)} = 1.08 \times \text{SCFM} \times \Delta \text{ Temperature}$

